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AN EMPIRICAL STUDY OF SOME CRITICAL MASS DATA<br>By<br>C. L. SCHUSKE and J.W. MORFITT

Y. 12 PLANT

CARBIDE AND CARBON CHEMICALS CORPORATION OAK RIDGE, TENNESEEE

# CARBIDE AND CARBON CHEMICALS CORPORATION 

> Y-12 PLANT

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SUPERINTENDENT 'S DEPARTMENT
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AN EMPIRICAL STUDY OF SOME CRITICAL MASS DATA

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## ABSTRACT

A simple empirical equation has been found which relates the critical height of a water-enclosed stainless steel reactor at a given moderation to its diameter. Certain empirical constants appearing in the relation have a simple physical interpretation which succeeds to a limited extent in bridging the gap between the experimental results with finite cylindrical reactors and Grueling'si theoretical treatment which is limited to infinite cylinders or slabs. The report also discusses certain comparisons between theoretical and experimental results.

Oak Ridge, Tennessee
December 6, 1949

1. Grueling, E., "Theory of Water-Tamped Water Boiler", LA-399, September 27, 1945.

## AN EMPIRICAL STUDY OF SOME CRITICAL MASS DATA

## INTRODUCTION

The results of a series of critical mass experiments performed in cylindrical geometry and utilizing aqueous solutions of uranyl fluoride of $93.4 \% U_{235}$ isotopic assay have been reported in "Critical Mass Studies, Part III". The application of curve fitting methods to certain of the empirical data there presented has a three-fold purpose.

1. To express the results of Part III in a more compact form and to make interpolation of data a more convenient process.
2. To allow estimation of critical conditions in a region beyond the range of the experimental data.
3. To provide a means for estimating the critical diameters of cylinders of infinite length and the critical thickness of slabs of infinite surface area for comparison with the critical dimensions obtained by the integral theory of Grueling. 3

In addition this particular approach may, in time, provide simpler and more direct mathematical treatments than currently exist in regard to certain basic problems.

PROCEDURE AND RESUTIS

The curves of Figure 1 were obtained by plotting the experimentally determined critical height of a cylinder as a function of its diameter for
2. Beck, C. D., A. D. Callihan, J.W. Morfitt, R. L. Murray, "Critical Mass Studies, Part III", Carbide and Carbon Chemicals Corporation, K-343, April 19, 1949. (This report will hereafter be referred as. Part III.)
3. Grueling, E., "Theory of Water-Tamped Water Boiler", LA-399, September 27, 1945.
various moderations in the manner given in Part III. Simpio physical

considerations, confirmed by experiment, show that each curve exhibits two asymptotes which are readily interpreted. The vertical asymptote, $D=$ constant, represents the diameter of a just critical infinite cylinder at the given moderation. Similarly, the horizontal asymptote, $H=$ constant, represents the height of a just critical infinite slab. There is a pair of asymptotes for each curve corresponding to a different H: $U_{235}$ ratio.

These data were submitted to various curve fitting teckniques and the most satisfactory fit was found to be a hyperbola of the form ( $D-a$ ) (H-b) $=c$ where $a$ and $b$ are the respective asymptotes interpreted above. The method by which the empirical equations were obtained from the distribution of experimental points follows a procedure described by Running. ${ }^{4}$ Its application to this problem is worked out in detail in the Appendix. By the method there described values of $a, b$ and $c$ can be readily obtained for each H: $U_{235}$ ratio, and are listed in Table I.
4. Running, Theodore R., "Empirical Formulas", John Wiley \& Sons, New York,
New York.

TABLE I
Iimiting Dimensions of Infinite Slab and Cyinder at Various H：T：こう Faミios

|  | Height of Infinite SIab（cm） | Diameter of Infinite Cylinder（cm） |  |
| :---: | :---: | :---: | :---: |
| H：U ${ }_{2} 35$ | b | a | c |
| 31.6 | 6.42 | 14.22 | 99.0 |
| 50.0 | 5.85 | 14.10 | 102.6 |
| 100.0 | 6.24 | 14.58 | 104．1 |
| 200.0 | 7.60 | 16.10 | 104．9 |
| 300.0 | 9.25 | 17.88 | 108.4 |
| 400.0 | 11.54 | 20.02 | 204.1 |
|  |  | Average | 104．0． |

Figure 2 is a plot of the data of Table $I$ ，i．e．，the thicimess of a just critical infinite slab（b）ana the diameter of a just criさiaai infinite cylinder（a）as a function of H： $\mathbb{U}_{235}$ ratio．Having obtaiced the necessary constants，one can then use the formula（ $D-a)\{H-b$ ）$=0$ to salcu－ late the critical height as a function of resctor diameter and moderation． The values so obtained when compared with the experimental vaiues provide a test of the fit of the empirical equation．Such a comparison is made it columns 3 and 4 in Table III beiow，which gives the calcuiated and experi－ mentally obtained vaiuss of the critical height for varioue size resctors under various conditions of moderation．It will be noted that the sgrees ment is well within the reproducibility of the experimentai data，and that the value of $c$ varies only slightly with $H: U_{235}$ ratio．There is not suf－ ficient evidence to indicate whether the variation in $c$ is random or is significant．The magnitude of the inherent inaccuracies in the experi－ mental data preclude the possibility of further refining the empirical technique．If $c$ is allowed to vary in the manner shown in Teble I its

calculation for the intermediate moderations becomes inconvenient. An average value of $c$ for ail moderations was used to simplify computation. Using this average value of $c$, together with the original experimental data, vaiues of $a$ and $b$ were recalculated to give the best fit under these conditions. The new values of a and $b$ were within about two millimeters of the originally obtained values as shown below in Table II. The assumption that $c=$ constant for all H:U 235 ratios implies that all curves shown in Figure 1 have same shape and differ only in the location of the asymptotes.

TABIE II
Iimiting Dimensions of Infinite Slabs and Cylinder Using Average Value of $c=104$

| H:U235 | Eeight of <br> Infinite Slat ( am) | Diameter <br> Infinite Cyli |
| :---: | :---: | :---: |
| Ratio | $\underline{b}$ | a |
| 30 | 6.70 | 14.1 |
| 50 | 6.12 | 14.0 |
| 100 | 6.45 | 14.5 |
| 150 | 7.00 | 15.4 |
| 200 | 7.70 | 16.2 |
| 250 | 8.55 | 17.0 |
| 300 | 9.47 | 17.9 |
| 350 | 10.50 | 18.9 |
| 400 | 11.65 | 20.1 |

It is interesting to note that the minimum diameter of a just critical infinite cylinder is slightly over 5.5 inches.

Using the data of Table II, the critical heights were recalculated for these somewhat simplified conditions and are reproduced in column 5 of Table III below:

Comparison of Calculated and Experimental Values of Critical Height
Critical Height

| $\begin{aligned} & \text { Reactor } \\ & \text { Diameter } \\ & \text { (inches) } \end{aligned}$ | $\begin{aligned} & \text { H:U } U_{235} \\ & \text { Ratio } \end{aligned}$ | $\begin{gathered} \text { From Part III } \\ (\mathrm{cm}) \\ \hline \end{gathered}$ | Using Values of $c$ from Table I ( cm ) | $\begin{gathered} \text { Using Average } \\ \text { Value }(\mathrm{c}=104) \\ (\mathrm{cm}) \\ \hline \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: |
| 15 | 50.0 | 10.1 | 10.1 | 10.0 |
|  | 100 | 10.6 | 10.6 | 10.4 |
|  | 200 | 12.5 | 12.4 | 12.5 |
|  | 300 | 14.7 | 14.6 | 14.6 |
|  | 400 | 17.3 | 17.3 | 17.4 |
| 12 | 30.0 | 12.8 | 12.8 | 13.1 |
|  | 50.0 | 12.6 | 12.1 | 12.4 |
|  | 100 | 13.0 | 12.7 | 13.0 |
|  | 200 | 15.1 | 14.9 | 15.0 |
|  | 300 | 17.7 | 17.8 | 17.8 |
|  | 400 | 21.5 | 21.4 | 21.7 |
| 10 | 30.0 | 15.4 | 15.6 | 15.9 |
|  | 50.0 | 15.0 | 14.9 | 15.3 |
|  | 100 | 15.5 | 15.8 | 16.1 |
|  | 200 | 18.8 | 19.0 | 19.1 |
|  | 300 | 23.7 | 23.7 | 23.4 |
|  | 400 | --- | 30.7 | 31.2 |
| 9 | 30.0 | 18.3 | 18.2 | 18.6 |
|  | 50.0 | 17.8 | 17.6 | 17.9 |
|  | 100. | 18.9 | 18.7 | 19.1 |
|  | 200 | 23.4 | 23.4 | 23.5 |
|  | 300 | 31.2 | 31.0 | 30.5 |
|  | 400 | --- | 47.8 | 48.9 |
| 8 | 30.0 | 23.0 | 22.9 | 23.5 |
|  | 50.0 | 22.2 | 22.3 | 22.6 |
|  | 100 | 24.4 | 24.2 | 24.8 |
|  | 200 | 32.8 | 32.9 | 33.3 |
|  | 300 | 53.6 | 5.3 .7 | 52.6 |
| 7 | 30.0 | 34.2 | 34.1 | 35.2 |
|  | 50.0 | 32.4 | 33.7 | 33.8 |
|  | 100 | 37.8 | 38.5 | 37.8 |
|  | 200 | --- | 73.2 | 76.0 |
| $6 \frac{1}{2}$ | 30.0 | 48.6 | 49.4 | 50.0 |
|  | 50.0 | 47.1 | 48.4 | 47.9 |
|  | 100 | 62.6 | 59.8 | 62.6 |

Comparing the simplified calculated values in column 5 with the experimental values of column 3, it is found that the average deviation from the experimental values is $1.4 \%$ with a maximum deviation of $5 \%$, well within the expected experimental error.

The critical dimensions having been obtainod, the mass may be calculated in the usual way. Figure 3 is a plot of the critical mass so obtained as a function of the $H: U_{235}$ ratio for a reactor eight inches in diameter, and illustrates that the general shape of such a curve agrees with its experimental counterpart in Part III and in addition provides a semi-quantitative idea of the steepness of the curve beyond the minimum. The experimental points have been plotted for comparison.

The same mothod may be applied to calculate the minimum critical mass in cylindrical geometry at a given moderation irrespective of cylinder dimensions. For any group of cylinders having the same volume the minimum surface area occurs for that cylinder where $H=D$. If it is assumed that the leakage in a critical water-enclosed cylinder is proportional to the surface area, the equilateral cylinder will have the smallest leakage and hence will noed the smallest mass to make it critical. Following this reasoning the minimum critical masses was calculated as a function of the moderation on the assumption that $H=D$ for such cylinders. Elementary theory predicts $H=.92 D$ for a reactor without an external water reflector. The ratio of $\mathrm{H}: \mathrm{D}$ would be expected to be still closer to unity for the reflected case. Since the expected deviation of $H / D$ from unit is of the order of the experimental error, the assumption that $H=D$ for minimum critical mass is satisfactory.


The critical mass was therefore calculated at various moderations for the case $H=D$ and the results plotted in Figure 4. A curve showing the theoretically calculated diameter of the critical water-reflected sphere as a function of moderation has been included for comparison. A sample calculation is given in the appendix. The broad minimum which extends from $H: U_{235}$ of 200 to $H: U_{235}$ of 800 is partially explained by reference to Figure 5. The latter consists of two relationships, (1) minimm critical volume and (2) grams of $U_{235}$ per unit volumn, both plotted as functions of the moderation. The critical mass curve is obtained by multiplying the ordinates of the two curres together for each value of the moderation. The fact that the critical mass is the product of an increasing function and a decreasing one accounts for the flainess of the minimum.

## COMPARISON WITH THEORY

An added advantage in having obtained the limiting dimensions of the infinite slab and cylinder is that they can be compared with values obtained by theoretical methods, as for example, the integral method of Grueling. 5 The results reported there for infinite cylinders and slabs have been recalculated using the appropriate $H$ and $G$ functions with more recently reported values of the constants 6,7 (tabulated in. Table VII),

[^0]

$\stackrel{\rightharpoonup}{\omega}$
and neglecting the effect of displacement on the core functions. The limiting dimensions obtained through the use of LA-399 can then be compared with the results obtained by the empirical method described here. The comparison is tabulated in Table IV and a graphical comparison is given in Figures 6 and 7.

TABLE IV
Comparison of Empirically Determined Limitinz Slab and Cylinder Dimensions with those Obtained by Methods of IA-399

| H: $\mathrm{U}_{235}$ | Slab Height |  | Cylinder Diameter |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | $\begin{aligned} & \text { Grueling } \\ & (\mathrm{cm}) \\ & \hline \end{aligned}$ | Empirical <br> Stainless <br> Steel(cm) | $\begin{aligned} & \text { Grueling } \\ & (\mathrm{cm}) \end{aligned}$ | Empirical <br> Stainless <br> Steel(cm) | Empirical <br> Constants |
| 30 | 4.66 | 6.70 | 13.61 | 14.1 | 101 |
| 31.6 |  | 6.42 |  | 14.22 | 99 |
| 50 | 4.95 | 5.85 | 13.89 | 14.10 | 102.6 |
| 100 | 5.51 | 6.24 | 14.58 | 14.58 | 104.1 |
| 200 | 6.75 | 7.60 | 16.10 | 16.18 | 104.9 |
| 300 | 7.85 | 9.25 | 17.60 | 17.88 | 108.4 |
| 400 | 9.02 | 11.54 | 19.48 | 20.02 | 104.1 |
| 500 | 10.08 |  | 21.11 |  | 104 |

When the effect of displacement on the $H$ and functions is considered, the slab and cylinder dimensions are increased slightly. Three such points have been plotted on the infinite slab graph to illustrate the amount of increase. The effect of introducing the displacement into the theoretical computation in the amount required by Grueling ${ }^{8}$ is still not sufficient to predict an increase in the critical dimensions below an H: $U_{235}$ of 50 found empirically. It may therefore be that displacement
8. Grueling, E., "Theory of Water-Tamped Water Boiler", LA-399, September 27, 1945.


has a much larger effect than has been predicted.
It should be pointed out that the empirical treatment is based on data in stainless steel reactors while the theoretical treatment neglects wall effects. Sufficient data were available for an aluminum cylinder at a H:U $\mathrm{U}_{235}$ ratio of 55 (observed minimm critical height at this moderation) to calculate empirically the infinite slab and infinite cylinder values. These values along with the Grueling values appear in tabular form below:

## TABLE V

Infinite Slabs and CFlinders @ H:U $U_{235}=55$ for Aluminum Cylinders

Empirically
Theoretical Derived Values

| Infinite Slab, Height (cm) | 5.0 | 4.3 |
| :--- | :---: | ---: |
| Infinite Cylinder, Diameter (cm) | 13.95 | 13.8 |

13.95 13.8

The values for the cylinder diameters fall well within the experimental error for both aluminum and stainless reactors, but the deviation of the values for the slabs is significant. It is not kmown whether the discrepancy is due to inaccuracies in the water boiler treatment or whether it arises from insufficient data for cylinders of larger diameter.

In sumary, this report has attempted to derive a simple expression for the relationship between height at criticality and diameter in certain cylindrical geometries and to obtain, from this relationship, the limiting dimensions of infinite cylinders and slabs for comparison with results obtained by purely theoretical methods.

The authors are indebted to $\mathrm{Dr} . \mathrm{A} . \mathrm{D}$. Callihan of the $\mathrm{K}-25$
Laboratories for many helpful suggestions during the preparation of this report. The graphs appearing in this report were prepared by the Y-12 Engineering Department.

Approved:
$\frac{\text { C. E. Larson }}{\text { c. }}$

## Authors:


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APPENDIX
The empirical equation $(D-a)(H-b)=c$ was passed through a distribution of experimental points by the following curve fitting techniquea ${ }^{9}$

Statement of the Theorem:
If two variables $x$ and $y$ are so related that the points represented by $x-x_{k}$, and $\frac{x-x_{k}}{y-y_{k}}$ lie on a straight line, the relation between the variable can be expressed by the equation $(x-a)(y-b)=$ constant.

If

$$
(x-a)(y-b)=c
$$

Let

$$
\begin{aligned}
& x-x_{k}=X \\
& y-y_{k}=Y \\
& x=X ; x_{k} \\
& y=Y \nmid y_{k}
\end{aligned}
$$

(1) becomes

$$
\begin{equation*}
\left(X-x_{k}-a\right)\left(Y+y_{k}-b\right)=c \tag{2}
\end{equation*}
$$

multiplying and collecting terms

$$
\begin{equation*}
X Y+\left(x_{k}-a\right) Y+\left(y_{k}-b\right) X+\left(y_{k}-b\right)\left(x_{k}-a\right)=c \tag{3}
\end{equation*}
$$

$\operatorname{But}\left(y_{k}-b\right)\left(x_{k}-a\right)=c$

$$
\begin{equation*}
X Y t\left(x_{k}-a\right) Y+\left(y_{k}-b\right) X=0 \tag{4}
\end{equation*}
$$

Dividing by $Y\left(y_{k}-b\right)(4)$ becomes

$$
\begin{equation*}
x / Y=-\frac{X}{y_{k}-b}-\frac{\left(A_{k}-a\right)}{\left(y_{k}-b\right)} \tag{5}
\end{equation*}
$$

9. Running, Theodore R., "Empirical Formulas", John Wiley \& Sons, New York, New York.

The slope of this linear Iunction $X / Y$ as a function of $X$ is

$$
\begin{align*}
\text { slope } & =-\frac{1}{\overline{y_{k}-b}}  \tag{6}\\
\text { and the intercept } & =-\frac{\left(\overline{x_{k}}-a\right)}{\left(y_{k}-b\right)}
\end{align*}
$$

A sample calculation follows.

## Sample Calculation:

1. To pass the empirical equation $(D-a)(H-b)=0$ through a distribution of experimental points, critical height as a function of the cylinder diameter at a $H: U_{235}$ ratio of 31.6 .

Since the experimental values of the diameter are given in inches it is convenient to leave them in inches until the final value of the infinite sylinder is obtained.

| $\overline{7}$ |  |  | TABLE VI |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
| D" | 6.5 | 7.0 | 8.0 | 9.0 | 10.0 |
| H cms | 49.0 | 34.0 | 22.5 | 18.1 | 15.3 |
| X" | 0 | . 5 | 1.5 | 2.5 | 3.5 |
| Y cms | 0 | -15.0 | -26.4 | -30.9 | -33.7 |
| $\mathrm{X} / \mathrm{Y}$ | -- | - . 0333 | - . 0568 | 0809 | - . 1038 |

where $X=(D-6.5)$ and $Y=H-49.0$

$$
\begin{aligned}
& \text { Slope }=-\frac{1}{49.0-b}=.0235 \begin{array}{c}
\text { (obtained from Figure } 8 \text { and the } \\
\text { precering theoretical discussion) }
\end{array} \\
& b=6.42 \mathrm{cms} \text { where } \mathrm{b} \text { is the infinite slab thickess at a } \\
& \quad \begin{array}{l}
\text { moderation of } 31.6
\end{array} \\
& \text { Intercept }=\frac{-6.5-\mathrm{a}}{49.0-\mathrm{b}}=-.021
\end{aligned}
$$

Substituting for $b$, a then becomes equal to 5.61 inches of 14.2 cm where a is the diameter of the infinite cylinder.


SLOPE $=\frac{\Delta x / r}{\Delta X}$
FIGURE 8

This equation then becomes

$$
\begin{equation*}
(D-14.2)(\pi-3.42)=0 \tag{8}
\end{equation*}
$$

The constant $c$ is obtained by subatituting tise eritical oylinder dimensions for a given $E: U_{23 j}$, oqual to 31.6 in this sase, into the above equation as follows.

From taile VI, tre $6 . j$ inch cylinaier vas erioical when the height of solution was 49.0 om
$D=6.5 "$ or 16.51 cm
$H=40.0 \mathrm{~cm}$
$a=14.2 \mathrm{~cm}$
$b=5.42 \mathrm{~cm}$
Equation (8) becomes
$(26.5-14.2)(45.0-5.42)=0$
$c=98.0 \mathrm{~cm}^{2}$
This erossaure for determining o was applied to the data for $7^{\prime \prime}$, $8^{\prime \prime}$, $9^{\prime \prime}$ and $10 "$ cylinder or the same $H: U_{235}$, ratio in Table I. The constants a were then averegen. The average value is $0 \because 0$. The equation for an $\operatorname{la}_{235}=31.5$ then becomes

$$
(D-14.2)(E-6.42)=99
$$

These are values reporteci in Table I in the main body of the report. The steps describsi here vere repeatsi for various values of the H: U235 ratio.
2. To calculate the micimum critical mass of a reactor at a given H: $U_{235}$

$$
\begin{aligned}
& \text { H:U } U_{235} \text { ratio }=50 \\
& (D-a)(H-b)=104 \quad \text { (from Table II) }
\end{aligned}
$$

If $\quad H=D$,
then

$$
(D-6.12)(D-14.0)=104
$$

from which

$$
\begin{aligned}
& D=20.37 \mathrm{~cm} \text { or } 8.02^{\prime \prime} \\
& \text { Critical volume }=6.64 \mathrm{I} \\
& \text { Oritical mass }=.480 \times 6.54=3.18 \text { kilograms. }
\end{aligned}
$$

The calculations were repeated for other moderations and the resulting minimum critical masses are plotted against H: $\mathrm{U}_{235^{\circ}}$

TABIE VII
Physical Constants Used in the Grueling Calculations

| $\eta_{235}$ | $=2.50$ neutrons/fission |
| :--- | :--- |
| $\left(\sigma_{f}\right)_{235}$ | $=546$ barns |
| $\left(\sigma_{a}\right)_{235}$ | $=642$ barns |
| $\sigma_{a} / \sigma_{f}$ | $=1.18$ |
| $\nu_{235}=\eta / 1.18$ | $=2.12$ neutrons/absorption |
| $\left(\sigma_{a}\right)_{238}$ | $=2.56$ barns |
| $\left(\sigma_{a}\right)_{H}$ | $=0.32$ barns |
| $\left(\sigma_{a}\right)_{0}$ | $=.001$ barns |
| $\left(\sigma_{a}\right)_{F}$ | $=.01$ barns |
| $\left(\sigma_{\mathrm{s}}\right)_{235}$ | $=8.2$ barns |
| $\left(\sigma_{\mathrm{s}}\right)_{H}$ | $=21$ barns |
| $p$ | $=.934=$ isotopic purity |

The subscripte $f, a$, and s refer to fission, absorption and scattering, respectively.


[^0]:    5. Grueling, E., "Theory of Water-Tamped Water Boiler", IA-399, September 27, 1945.
    6. Way, K., "Description of an Average Fission", MonP-192, December 10, 1946.
    7. Way, K., G. Haines, "Thermal Neutron Cross Sections for Elemerts and Isotopes H-B1", AECD-2138, October 11, 1948.
